

UNIVERSITY OF SWAZILAND

FINAL EXAMINATION 2011/12

BSC IV

<u>TITLE OF PAPER</u>	:	FLUID DYNAMICS
<u>COURSE NUMBER</u>	:	M455
<u>TIME ALLOWED</u>	:	THREE (3) HOURS
<u>INSTRUCTIONS</u>	:	1. THIS PAPER CONSISTS OF <u>SEVEN</u> QUESTIONS. 2. ANSWER ANY <u>FIVE</u> QUESTIONS
<u>SPECIAL REQUIREMENTS</u>	:	NONE

THIS EXAMINATION PAPER SHOULD NOT BE OPENED UNTIL
PERMISSION HAS BEEN GRANTED BY THE INVIGILATOR.

USEFUL FORMULAE

The gradient of a function $\psi(r, \theta, z)$ in cylindrical coordinates is

$$\nabla\psi = \frac{\partial\psi}{\partial r}\hat{r} + \frac{1}{r}\frac{\partial\psi}{\partial\theta}\hat{\theta} + \frac{\partial\psi}{\partial z}\hat{k}$$

The divergence and curl of the vector field

$$\underline{v} = v_r\hat{r} + v_\theta\hat{\theta} + v_z\hat{k}$$

in cylindrical coordinates are

$$\nabla \cdot \underline{v} = \frac{1}{r} \left\{ \frac{\partial}{\partial r}(rv_r) + \frac{\partial}{\partial\theta}(v_\theta) + \frac{\partial}{\partial z}(rv_z) \right\}$$

and

$$\nabla \times \underline{v} = \frac{1}{r} \det \begin{bmatrix} \hat{r} & r\hat{\theta} & \hat{k} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial\theta} & \frac{\partial}{\partial z} \\ v_r & rv_\theta & v_z \end{bmatrix}$$

The divergence of a vector

$$\underline{v} = v_r\hat{r} + v_\lambda\hat{\lambda} + v_\theta\hat{\theta}$$

in spherical coordinates

$$\nabla \cdot \underline{v} = \frac{1}{r^2} \frac{\partial(r^2 v_r)}{\partial r} + \frac{1}{r \sin\theta} \frac{\partial v_\lambda}{\partial\lambda} + \frac{1}{r \sin\theta} \frac{\partial(\sin\theta v_\theta)}{\partial\theta}$$

The convective derivative and Laplacian in cylindrical coordinates are

$$\begin{aligned} \frac{D}{Dt} &= \frac{\partial}{\partial t} + v_r \frac{\partial}{\partial r} + \frac{v_\theta}{r} \frac{\partial}{\partial\theta} + v_z \frac{\partial}{\partial z} \\ \nabla^2 &= \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2}{\partial\theta^2} + \frac{\partial^2}{\partial z^2} \end{aligned}$$

Identities

$$\begin{aligned} \underline{v} \cdot \nabla \underline{v} &= \nabla \left(\frac{v^2}{2} \right) - \underline{v} \times \underline{\omega} \\ \nabla \times (\nabla \times \underline{a}) &= \nabla \nabla \cdot \underline{a} - \nabla^2 \underline{a} \end{aligned}$$

QUESTION 1

- (a) Describe density of the air at a point. [4]
- (b) For the Lagrangian method describe
- (i) trajectory, (ii) velocity, (iii) acceleration. [2,2,2]
- (c) A velocity field is specified as
- $$\vec{V} = ax^2\vec{i} + bxy\vec{j}, \text{ where } a = 2/(m.s), b = -4/(m.s), \text{ and the coordinates are measured in meters.}$$
- (i) Is the flow field one-, two-, or three- dimensional? Explain.
- (ii) Calculate the velocity components at the point $(2, \frac{1}{2}, 0)$.
- (iii) Develop an equation for the streamline passing through this point. [1,1,3]
- (d) Derive the formula for convective derivative of the density. [5]

QUESTION 2

- (a) Which of the following sets of equations represent possible two-dimensional incompressible flow cases? Explain.
- (i) $u = x + y, v = x - y$;
- (ii) $u = x + 2y, v = x^2 - y^2$;
- (iii) $u = xt + 2y, v = x^2 - yt^2$. [3]
- (b) The stream function for a certain incompressible flow is given as $\psi = Axy$, A is a constant.
- (i) Plot several stream lines, including $\psi = 0$,
- (ii) Obtain an expression for the velocity field [2,3]
- (c) (i) Prove that $\vec{V} = \nabla\psi + \vec{k}$ and thus
- (ii) Show $v_r = \frac{1}{r} \frac{\partial\psi}{\partial\theta}, v_\theta = -\frac{\partial\psi}{\partial r}$. [3,2]
- (d) The stream function for a certain incompressible flow is given as $\psi(r, \theta) = -Ur \sin\theta + \frac{q\theta}{2\pi}$, where U represents the free stream velocity.
- (i) obtain an expression for the velocity field,
- (ii) find the stagnation points, and
- (iii) show that $\psi = 0$ there. [3,2,2]

QUESTION 3

(a) Consider a piston-cylinder apparatus. At one instant when the piston is $L_0 = 0.15m$ away from the closed end of the cylinder, the gas density is uniform at $\rho = 18kg/m^3$ and the piston begins to move away from the closed end at $V = 12m/s$. The gas motion is one-dimensional and proportional to the distance from the closed end; it varies linearly from zero at the end to $u = V$ at the piston.

(i) Evaluate the rate of change of gas density at this instant.

(ii) Obtain an expression for the average density as a function of time.

HINT: Density ρ is independent of x .

[5,6]

(b) Let $\bar{V} = \frac{m}{r^2} \bar{e}_r$ in spherical coordinates.

(i) Show that \bar{V} satisfies the continuity equation for incompressible flow, except at the origin O .

(ii) Let O be lye on a smooth surface S . What is the volume flow rate through S ?

[4,5]

QUESTION 4

a) Evaluate the circulation of a line vortex.

[3]

b) The vorticity of a certain incompressible flow is given by the following formula

$$\omega = \begin{cases} -Ar \sin \theta, & \text{for } r < a \\ 0, & \text{for } r > a \end{cases}$$

Find the corresponding stream function.

[12]

HINT: $\nabla + (\nabla + \bar{A}) = \nabla \nabla \cdot \bar{A} - \nabla^2 \bar{A}$.

(c) A flow is represented by the velocity field

$$\bar{V} = 10x\bar{i} - 10y\bar{j} + 30\bar{k}.$$

Determine if the field is

(i) a possible incompressible flow,

(ii) irrotational.

[2,3]

QUESTION 5

a) A cylindrical container, partly filled with liquid, is rotated at a constant angular velocity ω about the vertical axis. After a short time there is no relative motion, the liquid rotates as a rigid body. Using the Euler equation determine the shape of the free surface if the radius of cylinder is R and the original surface height in the absence of rotation is h_0 . [12]

b) The velocity distribution for laminar flow between fixed parallel plates is given by

$$u = u_{max} \left[1 - \left(\frac{2y}{h} \right)^2 \right],$$

where h is the distance separating the plates and the origin is placed midway between the plates.

Consider $\mu = 1.1 \times 10^{-3} \text{ kg/ms}$, $u_{max} = 0.3 \text{ m/s}$, $h = 0.5 \text{ mm}$. Calculate

(i) the shear stress on the upper plate and give direction,

(ii) the force on a 0.5 m^2 section of the upper plate and give its direction. [4,2]

(c) Write Navier-Stokes equation for incompressible flow. [2]

QUESTION 6

a) An incompressible flow field is given by

$$\vec{V} = (Ax + By)\vec{i} - Ay\vec{j}, \text{ where } A = 1 \text{ s}^{-2}, \quad B = 2 \text{ s}^{-2}, \text{ the coordinates are in meters.}$$

(i) Find the acceleration of a fluid particle at point $(x, y) = (1, 2)$.

(ii) Find the pressure gradient at the same point, if $\vec{g} = -g\vec{j}$, and fluid is water,

$$\rho = 1000 \text{ kg/m}^3. \quad [6,4]$$

b) The plane $y = 0$ oscillates so that its velocity is in the plane $y = 0$ and has magnitude $V \cos \omega t$, where V and ω are constants. Above the plane there is viscous incompressible fluid. Body forces are negligible.

(i) Show that

$$\frac{\partial u}{\partial t} = \gamma \frac{\partial^2 u}{\partial y^2},$$

(ii) Separate variables to show that

$$u(y, t) = \text{Re} \left\{ V \exp \left(i\omega t - \sqrt{\frac{i\omega}{\gamma}} y \right) \right\} \quad [5,5]$$

QUESTION 7

- a) Derive the mass conservation equation for incompressible flow in dimensionless form. [4]
- b) (i) Define similar flow, and
- (ii) Explain how the idea of similarity is used to design the experimental models. [2,2]
- c) Water flows steadily up a vertical 0.1m diameter pipe and out the nozzle, which is 0.05m in diameter, discharging to atmospheric pressure. The stream velocity at the nozzle exit must be 20m/s. The flow is frictionless. The pipe is 4m long.
- (i) Write Bernoulli's equation.
- (ii) find velocity at inlet,
- (iii) calculate the gage pressure required at inlet. [2,3,7]