UNIVERSITY OF SWAZILAND

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FINAL EXAMINATION 2012/13

BSC IV

TITLE OF PAPER	:	FLUID DYNAMICS
COURSE NUMBER	:	M455
TIME ALLOWED	:	THREE (3) HOURS
INSTRUCTIONS	:	1. THIS PAPER CONSISTS OF <u>SEVEN</u> QUESTIONS.
SPECIAL REQUIREMENTS	•	2. ANSWER ANY <u>FIVE</u> QUESTIONS NONE

THIS EXAMINATION PAPER SHOULD NOT BE OPENED UNTIL PERMISSION HAS BEEN GRANTED BY THE INVIGILATOR.

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USEFUL FORMULAE

The gradient of a function $\psi(r, \theta, z)$ in cylindrical coordinates is

$$abla \psi = rac{\partial \psi}{\partial r} \hat{r} + rac{1}{r} rac{\partial \phi}{\partial heta} \hat{ heta} + rac{\partial \psi}{\partial z} \hat{k}$$

The divergence and curl of the vector field

$$\underline{v} = v_r \hat{r} + v_\theta \hat{\theta} + v_z \hat{k}$$

in cylindrical coordinates are

$$\nabla \cdot \underline{v} = \frac{1}{r} \left\{ \frac{\partial}{\partial r} (rv_r) + \frac{\partial}{\partial \theta} (v_\theta) + \frac{\partial}{\partial z} (rv_z) \right\}$$

and

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$$\nabla \times \underline{v} = \frac{1}{r} \det \begin{bmatrix} \hat{r} & r\hat{\theta} & \hat{k} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial \theta} & \frac{\partial}{\partial z} \\ v_r & rv_\theta & v_z \end{bmatrix}$$

The divergence of a vector

$$\underline{v} = v_r \hat{r} + v_\lambda \hat{\lambda} + v_\theta \hat{\theta}$$

in spherical coordinates

$$\nabla \cdot \underline{v} = \frac{1}{r^2} \frac{\partial (r^2 v_r)}{\partial r} + \frac{1}{r \sin \theta} \frac{\partial v_\lambda}{\partial \lambda} + \frac{1}{r \sin \theta} \frac{\partial (\sin \theta \, v_\theta)}{\partial \theta}$$

The convective derivative and Laplacian in cylindrical coordinates are

$$\frac{D}{Dt} = \frac{\partial}{\partial t} + v_r \frac{\partial}{\partial r} + \frac{v_\theta}{r} \frac{\partial}{\partial \theta} + v_z \frac{\partial}{\partial z}$$
$$\nabla^2 = \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2}$$

Identities

$$\underline{v} \cdot \nabla \underline{v} = \nabla \left(\frac{v^2}{2}\right) - \underline{v} \times \underline{\omega}$$
$$\nabla \times (\nabla \times \underline{a}) = \nabla \nabla \cdot \underline{a} - \nabla^2 \underline{a}$$

(a) Define a particle path.	[4]
(b) Describe the Eulerian method of treating motion of continuous medium.	[4]
(c) A velocity field is specified as $\overline{V} = Axy\overline{i} + By^2\overline{j}$, where $A = 1m^{-1}s^{-1}$,	
$B = -\frac{1}{2} m^{-1} s^{-1}$, and the coordinates are measured in meters.	
(i) Find the dimension of flow fluid. Explane.	
(ii) Calculate the velocity components at the point $(1, 2, 0)$.	
(iii) Develop an equation for the streamline passing through this point	[1,1,3]
(d) (i) Derive the formular for convective derivative of the density.	
(ii) What is a physical meaning of convective derivative of velocity?	[5,2]

QUESTION 2

- (a) The three components of velocity are given by u = Ax + By + Cz,
- V = Dx + Ey + Fz and w = Gx + Hy + Jz.

Determine the relationship among the coefficients A through J that is necessary if this is to be a possible incompressible flow. [3]

- (b) Consider two-dimensional incompressbile flow.
- (i) Define the stream function $\psi(x, y)$,

(ii) Show that
$$\psi(x, y) = \int_{b}^{y} u(x, \eta) d\eta - \int_{a}^{x} U(\xi, b) d\xi$$

where a and b are constants. [1,4]

(c) Determine the family of stream functions that will yield the velocity field

$$\overline{V} = (x^2 - y^2)\overline{i} - 2xy\overline{j}.$$
[5]

(d) Incompressible flow around a circular cylinder of radius a has a stream function

$$\psi(r,\theta) = -Ur\sin\theta + \frac{Ua^2\sin\theta}{r}, \quad U \text{ is constant.}$$

- (i) obtain an expression for the velocity field,
- (ii) find V_r along the circle r = a,
- (iii) Locate the points along r = a where $|\overline{V}| = U$. [3,2,2]

(a) Consider a piston-cylinder apparatus. At one instant when the piston is L_0 away from the closed end of the cylinder the gas density is uniform at $\rho = \rho_0$, and the piston begins to move away from the closed end at $V = V_0$. The gas velocity is one-dimensional and proportional to the distance from the closed end; it varies linearly from zero at the end to $U = V_0$ at the piston.

(i) Show that
$$\frac{d\rho}{dt}\Big|_{t=0} = -\rho_0 \frac{V_0}{L_0}$$
,

(ii) Find $\rho(t)$.

HINT: Density ρ is independent of x.

[5,5]

[3]

(b) Water flows along a pipe whose area of cross-section A(x) varies slowly with the coordinates x along the pipe. Flow is almost in x-direction.

(i) Explain average velocity over cross-section

(ii) What is Eulerian acceleration?

(iii) Show that Lagrange acceleration is
$$a_L = -\frac{Q^2}{A^3} \frac{dA}{dx}$$
, where Q is a volume flow rate. [2,1,4]

(c) Prove $\overline{V} = \nabla + \psi \overline{k}$ in the usual notations.

a) Consider a so-called "rigid-body rotation" model: $V_r = 0$, $V_{\theta} = Ar$, where A is a constant. Find stream function. [3]

b) Consider the Rankine's vortex

$$arpi = \left\{ egin{array}{ccc} \Omega \overline{k}, & for & r < a \ 0, & for & r > a \end{array}
ight.$$

(i) Find the stream function,

(ii) Find velocity V_{θ} .

ş.,

HINT: $\nabla + (\nabla + \overline{A}) = \nabla \nabla \cdot \overline{A} - \nabla^2 \overline{A}.$

(c) A flow is respresented by the velocity field

$$\overline{V} = (4x^2 + 3y)\overline{i} + (3x - 2y)\overline{j}$$

Determine if the field is

(i) a possible incompressible flow,

(ii) irrational.

 $^{[2,3]}$

[7,5]

a) Describe the stress components σ_n and τ_n .

[4]

[5]

[2]

[3]

b) Consider liquid of constant density ρ in the field of gravity. Show that

$$p = p_0 - \rho g z$$

in the usual notations.

c) Define the Newtonian fluid.

d) The velocity distribution for laminar flow between fixed parallel plates is given by

$$u = u_{max} \left[1 - \left(\frac{2y}{h} \right)^2 \right],$$

where h is the distance separating the plates, and the origin is placed midway between the plates. Consider $\mu = 1.1 \times 10^{-3}$ kg/ms, $u_{max} = 0.05m/s$, h = 5mm. Calculate

(i) the shear stress on the lower plate and give direction,

- (ii) the force on a $0.3m^2$ section of the lower plate and give its direction [4,2]
- e) Write Navier-Stokes equation for inviscid flow.

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a) A velocity field in a fluid with density of $1500 kg/m^3$ is given by

$$\overline{V} = (Ax - By)t\overline{i} - (Ay + Bx)t\overline{j},$$

where $A = 1s^{-2}$, $B = 2s^{-2}$, x and y are meters, and t in seconds. Body and viscous forces are negligible.

(i) Find the acceleration of a field particle at point (x, y) = (1, 2) and t = 1 sec.

(ii) Find the pressure gradient at the same point and same time.

b) Consider steady viscous incompressible flow between two stationery plates located at y = 0 and y = 1. Given that pressure at x = 0 and x = L is p_o and p_L respectively, with $p_0 > p_L$. The effect of body forces is negligible. Put $\overline{V} = u(x, y)\overline{i}$ and show that

[6,2]

[2,2]

(i)
$$\frac{\partial p}{\partial x} = \mu \frac{\partial^2 u}{\partial y^2},$$

(ii) $p(x) = p_0 - \frac{p_0 - p_L}{L}x,$
(iii) $u(y) = y(1-y)\frac{p_0 - p_L}{2\mu L}.$
[5,4,3]

QUESTION 7

a) Derive the Navier-Stokes equation in dimensionless form introducing the characteristic length and velocity. [6]

b) (i) Define Reynolds number,

(ii) Find dimension of Reynolds number.

c) Water flow in a circular pipe. At one section the diameter is 0.3m, the static pressure is 260kpa (gage), the velocity is 3m/s, and the elevation is 10m above ground level. At a section downstream at ground level the pipe diameter is 0.15m. Friction effects may be neglected.

- (i) Write Bernoulli's equation,
- (ii) Find velocity at downstream section,
- (iii) Calculate the gage pressure at the downstream section. [2,3,7]